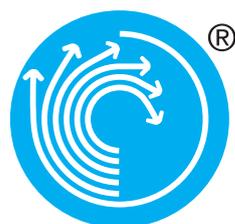


TE

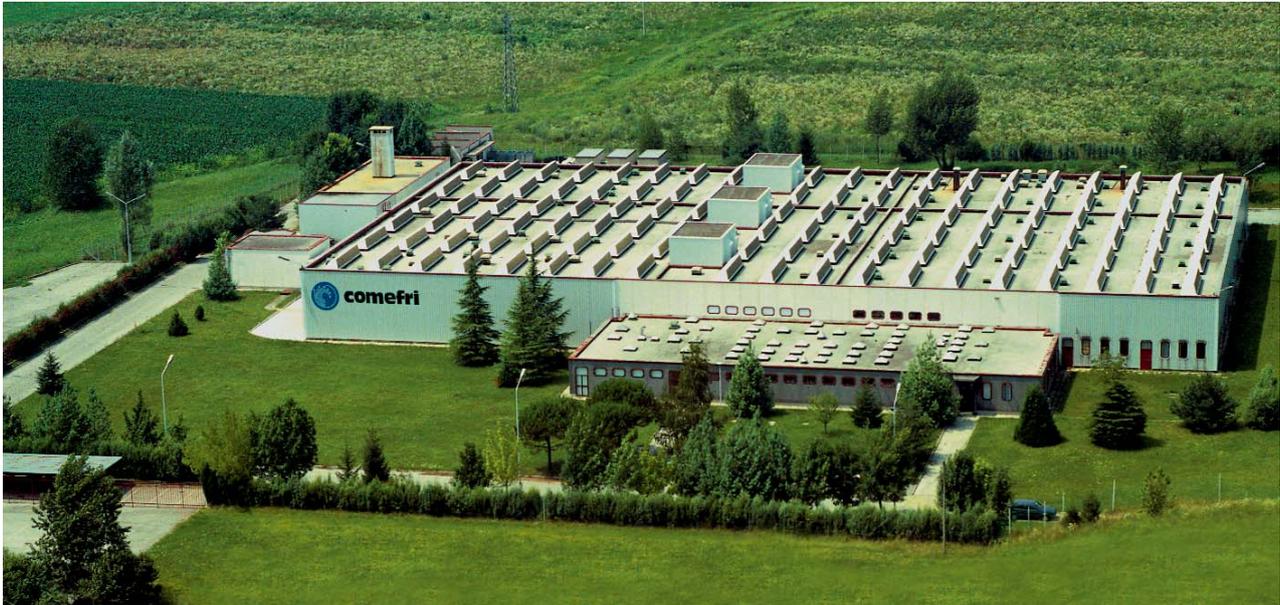
**GLASS REINFORCED POLYAMID
WHEELS FOR PLENUM FANS**



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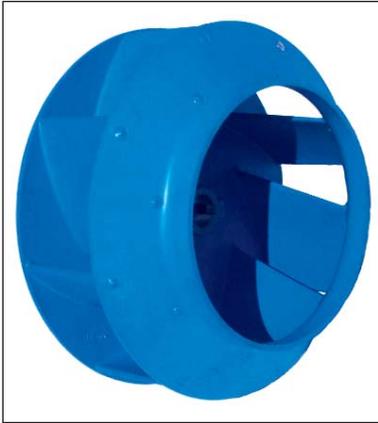
COMEFRI SpA factory at Magnano in Riviera (UD) - Italy with 14,500 m² manufacturing floor space, which produces radial fans for HVAC products.



COMEFRI SpA factory at Artegna (UD) - Italy with 6,300 m² manufacturing and laboratory floor space for the production of standard and special application industrial fan.



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1. GENERAL DESCRIPTION

The single inlet glass reinforced polyamid centrifugal wheel TE, has been designed for clean air. Due to its low weight and moment of inertia, the TE wheel is particularly suitable for direct drive applications.

A substantial aluminium hub is connected to the backplate. The hub bore is precision machined and incorporates a keyway and locking screw.

All impellers are statically and dynamically balanced to a grade of Q=6,3 in accordance with DIN ISO 1940-1 and ANSI S2.19 - 1989.

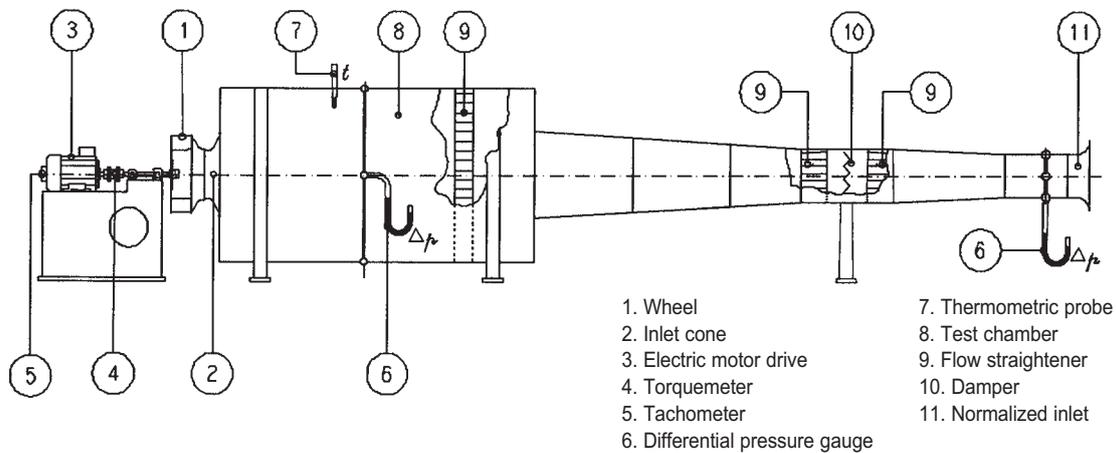
The inlet cone is designed to give the best aerodynamic performance. Maximum operating temperature 60°C for all the wheels.

2. WHEEL PERFORMANCE

2.1 Performance Data

The TE high performance airfoil impeller has been tested, in Comefri's certified laboratory, in accordance with DIN 24163 / BS 848 - Part.1 / ISO 5801 / AMCA 210 fig.14 (ref. to figure) and results refer at density of $\rho = 1,2 \text{ kg/m}^3$. Power ratings not include bearing losses.

Performance test rig scheme according to DIN 24163 / BS 848 Part.1 / ISO 5801 / AMCA 210 fig.14



- 1. Wheel
- 2. Inlet cone
- 3. Electric motor drive
- 4. Torquemeter
- 5. Tachometer
- 6. Differential pressure gauge
- 7. Thermometric probe
- 8. Test chamber
- 9. Flow straightener
- 10. Damper
- 11. Normalized inlet

The performance curves include the following information:

Static pressure	Δp_{stat}	[Pa]
Volume air flow	V	[m ³ /h]
Wheel absorbed shaft power	P_w	[kW]
Fan speed	n	[min ⁻¹]
Static Efficiency	$\eta_{st} = \frac{\Delta p_{stat} \cdot V}{P_w \cdot 36000}$	[%]
Total Efficiency	$(\eta_t)^*$	[%]
Sound Power Level	L_{w6}	[dB]

* theoretic value calculated taking into account the dynamic pressure at the impeller outlet

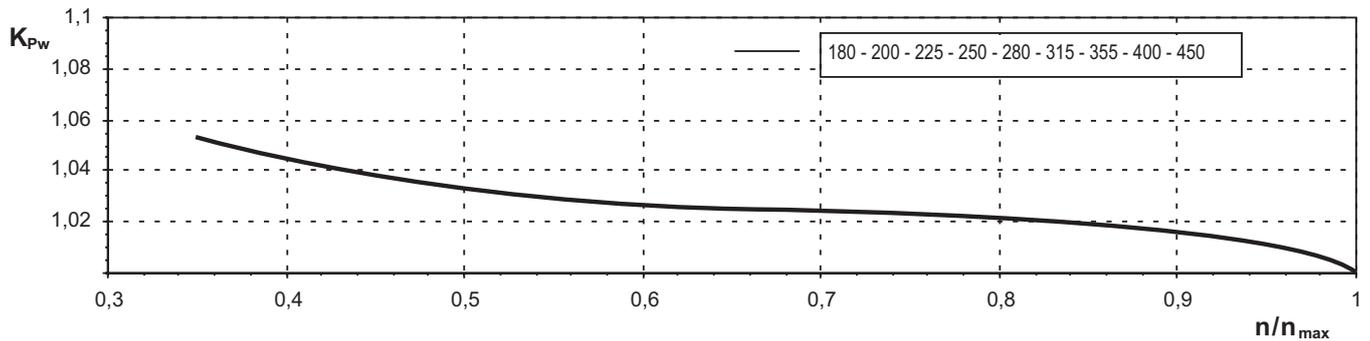


2.2. Power correction

It is well known that fan efficiency decreases when fan is running far from its maximum speed. The efficiencies marked in the performance graph curves must be corrected considering the real wheel speed n .

To obtain the correct absorbed power, multiply the readed value on performances curve by a corrective factor “ K_{Pw} ”, for the chosen wheel speed.

The factor “ K_{Pw} ” can be read off from the graph 2.2, as a function of the ratio n / n_{max} , where n is the required wheel speed and n_{max} is the maximum permissible wheel ($K_{Pw}=1$ when $n = n_{max}$).



Graph n° 2.2

2.3. Temperature and altitude correction factors

The performance charts refer to the standard air condition, i.e. $\rho = 1,2 \text{ kg/m}^3$, 20°C temperature and sea level. For different operating conditions the performance data must be corrected due to the change in air density.

Fan laws relate performance variables for any fan of a given design.

Static pressure varies directly as the ratio of the air densities, K_p

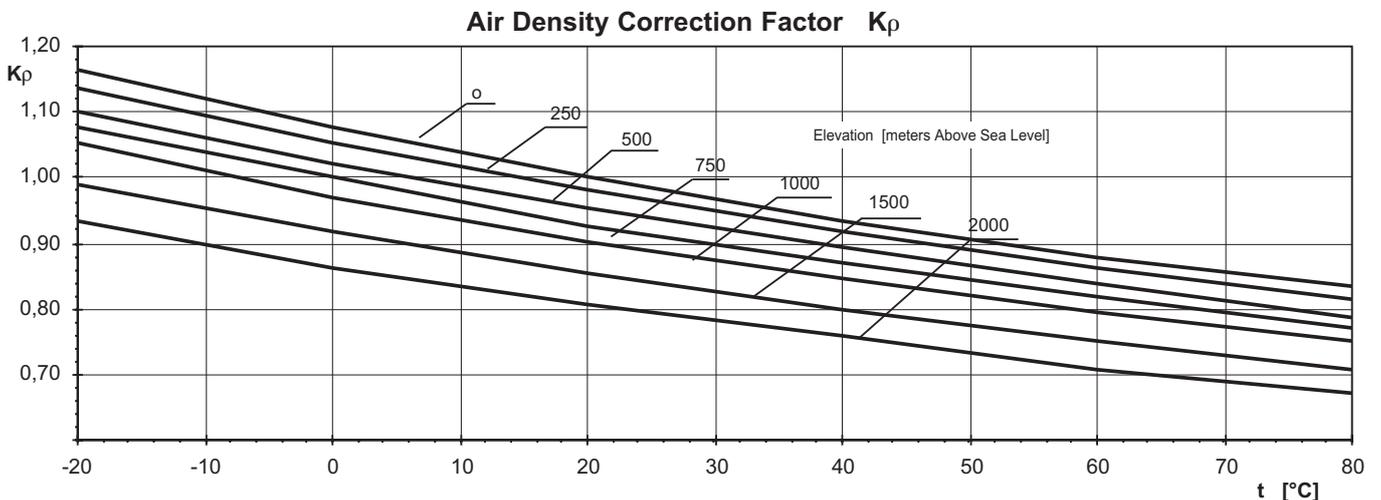
$$\Delta p_{stat2} = \Delta p_{stat1} \cdot K_p$$

Absorbed power varies directly as the ratio of the air densities, K_p

$$P_{w2} = P_{w1} \cdot K_p$$

The Graph 2.3 contains air density ratios K_p for temperatures from -20°C to 80°C and elevations up to 2000 m above sea level.

Please refer to the Selection Example 4.3., for further details on the correct selection procedure.



Graph n° 2.3



2.4 Motor selection

To determine the motor rating P_N , the wheel absorbed shaft power P_w must be increased by a factor f_w to accommodate for the arrangement losses, safety margins,.....etc.

$$P_N = P_w (1 + f_w)$$

The factor f_w can be chosen from the following figures:

$$\begin{array}{ll}
 P_w \leq 3 & k_w \dots f_w = 0,08 \\
 P_w > 3 & k_w \dots f_w = 0,06
 \end{array}$$

When selecting the suitable motor, the run-up time must be considered. The run-up time " t_a " can be calculated according to the following formula:

$$t_a = 8 \cdot \frac{J \cdot n^2}{P_N} \cdot 10^{-6}$$

where:

t_a	acceleration time	[s]
J	moment of inertia of the revolving parts	[kg m ²]
n	impeller revolution	[min ⁻¹]
P_N	motor rating	[kW]

If " t_a " exceed the motors' manufacturer recommendations, a larger motor or a high-torque type must be used.



3. SOUND LEVELS

The measurement of noise levels have been made according to ISO 3747, DIN 45635 and BS 848 Part.2, Survey Method using a Reference Sound Source, Standards, using a Bruel & Kjaer calibrated sound source and a real-time frequency analyzer.

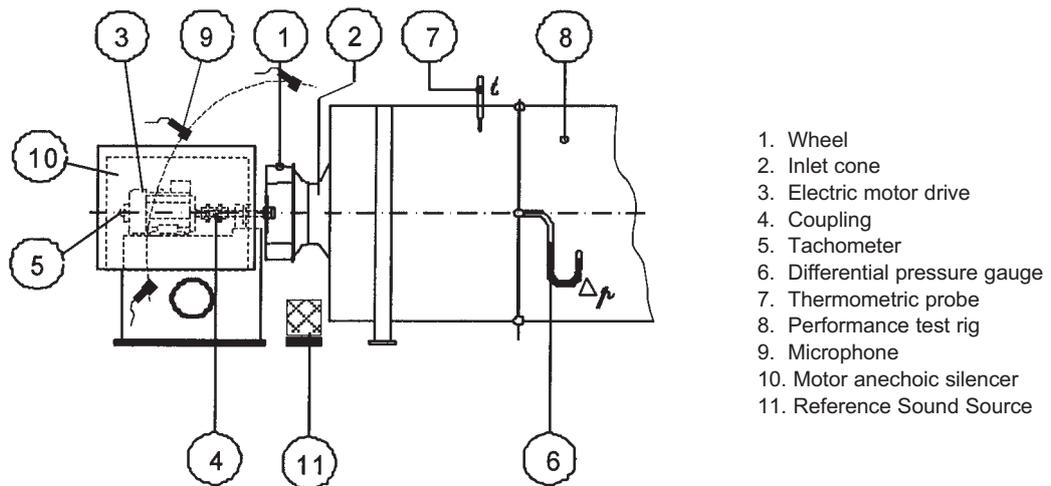
The Sound Power Level L_{w6} , referred to $W_0=10^{-12}$ Watt, required for calculation and design of sound attenuation units, are marked on the performance curves.

Symbols and Formulae:

L_{w6}	Total Sound Power Level at the wheel outlet	[dB]
L_{wocf6}	Sound Power Level at a specific Octave Band Mid-Frequency	[dB]
f_m	Octave Band Mid-Frequency	[Hz]
ΔL_{wocf6}	Difference between the Total Sound Power Level L_{wocf6} at the corresponding Octave Band Mid-Frequency and the Total Sound Power Level L_{w6}	[dB]
ΔL_{wA6}	Difference between the Total Sound Power Level L_{w6} and the A-weighted Total Sound Power Level L_{wA6}	[dB(A)]

Sound measurement test rig scheme according to ISO 3747 / DIN 45635 / BS 848 Part.2

Uncertainty in determining A-weighted Sound Power Level by the survey method is 4 dB(A)



Fan Sound Data are determined as follows:

1) The Total Sound Power Level L_{w6} , at the wheel outlet can be read from the Performance Chart, for a given wheel performance.

2) The Sound Power Level L_{wocf6} , at a specific Octave Band Mid-Frequency, at the wheel outlet, can be determined from following formula:

$$L_{wocf6} = L_{w6} + \Delta L_{wocf6}$$

3) The A-weighted Total Sound Power Level at the wheel outlet, can be determined from following formula:

$$L_{wA6} = L_{w6} - \Delta L_{wA6}$$

4) The values for ΔL_{wocf6} and ΔL_{wA6} for each wheel size can be found in the Sound Data Tables section 3.1., considering the range of wheel speed.



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**GLASS REINFORCED POLYAMID WHEELS
FOR PLENUM FANS - TE**

3.1. Sound Data tables

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 180	n < 4200	5,7	-7	-5	-8	-6	-12	-15	-20	-24
	n ≥ 4200	4,4	-7	-6	-9	-5	-10	-13	-18	-22

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 200	n < 4200	5,7	-7	-5	-8	-6	-12	-15	-20	-24
	n ≥ 4200	4,4	-7	-6	-9	-5	-10	-13	-18	-22

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 225	n < 4200	4,3	-7	-5	-9	-7	-12	-11	-13	-16
	n ≥ 4200	4,0	-7	-8	-7	-6	-10	-11	-14	-17

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 250	n < 3350	2,1	-10	-8	-7	-8	-7	-8	-12	-16
	n ≥ 3350	3,1	-6	-9	-10	-8	-11	-8	-12	-15

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 280	n < 3350	4,1	-9	-7	-6	-9	-10	-10	-15	-17
	n ≥ 3350	5,2	-5	-10	-8	-9	-12	-11	-15	-18

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 315	n < 2700	4,0	-5	-10	-7	-10	-10	-9	-15	-18
	n ≥ 2700	3,7	-7	-6	-8	-8	-10	-9	-15	-17

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 355	n < 2700	3,8	-5	-11	-7	-10	-10	-9	-14	-16
	n ≥ 2700	2,5	-7	-9	-9	-8	-10	-7	-12	-16

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 400	n < 2100	5,2	-3	-11	-9	-11	-9	-13	-15	-19
	n ≥ 2100	4,3	-4	-12	-9	-11	-9	-10	-15	-18

Fan Model and Size	Speed range	ΔL_{WA6}	ΔL_{woc6} 63	ΔL_{woc6} 125	ΔL_{woc6} 250	ΔL_{woc6} 500	ΔL_{woc6} 1000	ΔL_{woc6} 2000	ΔL_{woc6} 4000	ΔL_{woc6} 8000
TE 450	n < 2100	3,4	-8	-10	-6	-9	-8	-10	-14	-16
	n ≥ 2100	2,7	-8	-12	-6	-9	-8	-8	-14	-16

4. MINIMUM DISTANCES AND SELECTION CRITERIA

4.1. Minimum distances

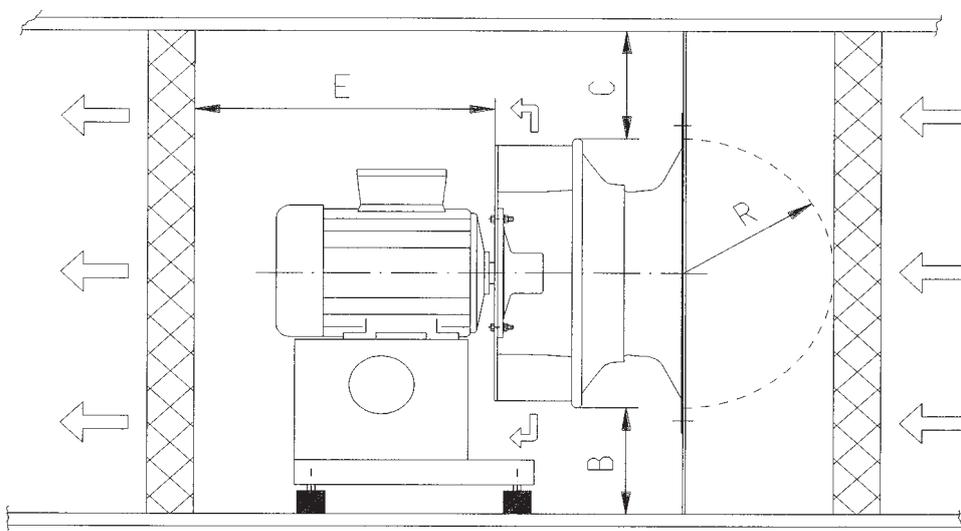
The following minimum distances are recommended for a correct plug fan installation:

$$E > \varnothing A$$

$$R > 0,5 \cdot \varnothing A$$

$$B, C > 0,4 \cdot \varnothing A$$

where $\varnothing A$ = max wheel diameter (see table).



WHEEL \varnothing	$\varnothing A$ [mm]
180	180
200	202
225	226
250	250
280	282
315	316
355	357
400	402
450	455

4.2. Plenum effect losses

The loss associated with the duct take off from the plenum must be added to the static pressure required by the fan.

These losses, as literature indicates, are:

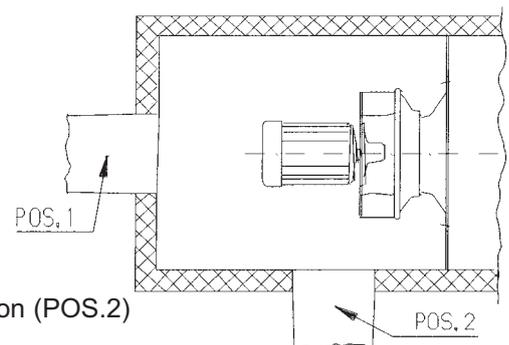
for axial ducted discharge (POS.1)

$$\Delta 1 = 2 \cdot p_{dc}$$

for radial ducted discharge (POS.2)

$$\Delta 2 = 1,5 \cdot p_{dc}$$

where: p_{dc} = dynamic pressure in the discharge duct



Example: 0,315 · 0,315 m discharge duct ($A = 0,099 \text{ m}^2$), radial position (POS.2)

$$\rho = \text{air density} = 1,2 \text{ kg/m}^3$$

$$V = \text{required air flow volume } 3600 \text{ m}^3/\text{h} = 1 \text{ m}^3/\text{s}$$

$$\Delta p_r = \text{required static pressure} = 1008 \text{ Pa}$$

$$p_{dc} = 1/2 \cdot \rho \cdot (V/A)^2 = 0,5 \cdot 1,2 \cdot (1/0,099)^2 = 61 \text{ Pa}$$

$$\Delta 2 = 1,5 \cdot 61 = 92 \text{ Pa}$$

TE selection has to be done with the following data:

$$\Delta p_{stat} = 1008 + 92 = 1100 \text{ Pa}$$

$$V = 3600 \text{ m}^3/\text{h}$$



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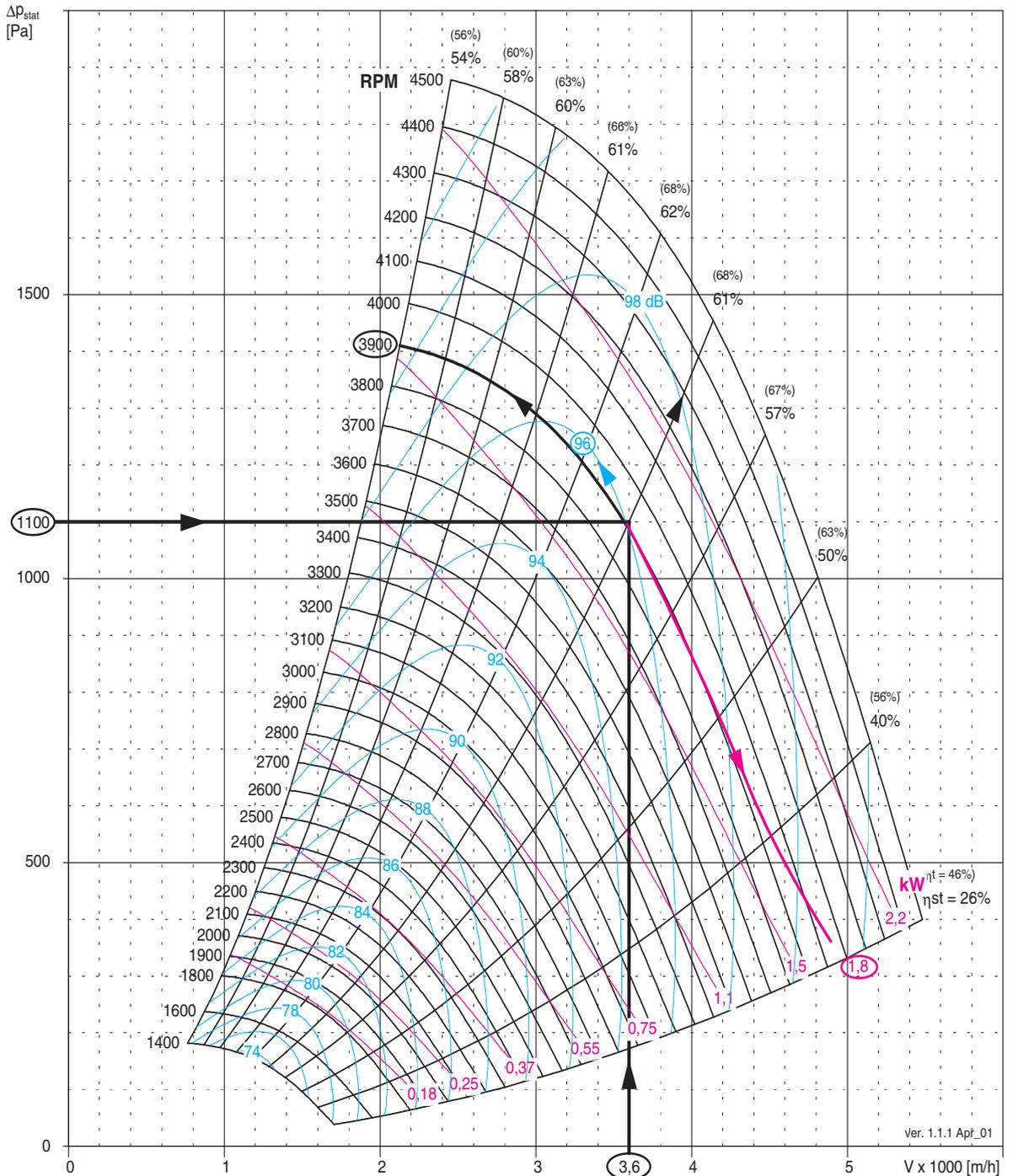
GLASS REINFORCED POLYAMID WHEELS FOR PLENUM FANS - TE

4.3. Selection Example

Wheel selection for the following operating parameters:

V = 3600 m³/h
 Δp_{stat} = 1100 Pa
 ρ = 1,2 kg/m³
 t = 20 °C

Model/Size	TE 280
Wheel Max RPM [min ⁻¹]	4400
Wheel Max kW	2,75
Wheel No. Blades	8



Selection model and size: **TE 280**

n = 3900 min⁻¹
 L_{w6} = 96 dB
 η_{st} = 61 %
 P_w = 1,8 kW
 η_t = 68 %



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**GLASS REINFORCED POLYAMID WHEELS
FOR PLENUM FANS - TE**

a) Power correction

The corrected absorbed power can be calculated as follow:

The ratio n/n_{max} is equal to: $3900 / 4400 = 0,886$

The value of K_{Pw} is read on the graph 2.2, corresponding to: $K_{Pw} = 1,015$

therefore the corrected absorbed power P_{wcorr} is: $P_{wcorr} = 1,8 \cdot 1,015 = 1,827 \text{ kW}$

b) Sound data

The following steps must be followed to determine the Octave Band values:

b1) Read on the Sound Data Table 3.1, for TE 280, each Octave Band and considering the selected wheel speed ($n \geq 3350 \text{ min}^{-1}$) the appropriate values for ΔL_{wocf6} :

ΔL_{wA6}	ΔL_{wocf6}							
5,2	-5	-10	-8	-9	-12	-11	-15	-18

b2) Apply these corrections to $L_{w6} = 96 \text{ dB}$ (add the ΔL_{wocf6} values) to obtain the values of L_{wocf6} :

| L_{wocf6} |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 63 | 125 | 25 | 500 | 1000 | 2000 | 4000 | 8000 | |
| 91 | 86 | 88 | 87 | 84 | 85 | 81 | 78 | |

b3) To obtain the L_{wA6} A-weighted Total Sound Power value, add to L_{w6} the ΔL_{wA6} value

$$L_{wA6} = L_{w6} - \Delta L_{wA6} = 96 \text{ dB} - 5,2 = 90,8 \text{ dB(A)} \text{ (rounded off to } 91 \text{ dB(A))}$$

b4) To obtain the A-Weighted Octave Band values, apply to each octave-band value the correction factor, listed here below:

Octave Band Mid-Frequency	63	125	250	500	1000	2000	4000	8000
A-Weighting	-26	-16	-9	-3	0	+1	+1	-1

(Values rounded off)

L_{wocfA6} , A-weighted values, are consequently $L_{wocfA6} = L_{wocf6} - (\text{A-weighting})$:

| L_{wocf6} |
|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|-------------|
| 63 | 125 | 25 | 500 | 1000 | 2000 | 4000 | 8000 | |
| 65 | 70 | 79 | 84 | 84 | 86 | 82 | 77 | |

c) Temperature and altitude correction

If temperature and altitude at which the fan will operate are not standard, the pressure value used for the selection must be corrected. Let's consider the following parameters:

Air volume: 3600 m³/h
Required static pressure: 935 Pa referred to the following conditions:
Operating temperature: 40 °C
Altitude: 1000 m a.s.l.

From K_p Air Density Correction Factor table (Graph 2.3) the value of 0,85 is obtained.

The corrected pressure, to be used for the selection on the performance chart, is therefore:

$$\Delta p_{\text{stat corr}} = \frac{\Delta p_{\text{stat}}}{K_p} = \frac{935}{0,85} = 1100 \text{ Pa}$$

Selection should be made with a corrected static pressure equal to 1100 Pa.

We obtain the following operating parameters:

Selected model and size:
TE 280
 $n = 3900 \text{ [min}^{-1}\text{]}$

Effective absorbed power on wheel shaft (corrected value) at that altitude and temperature, will be:

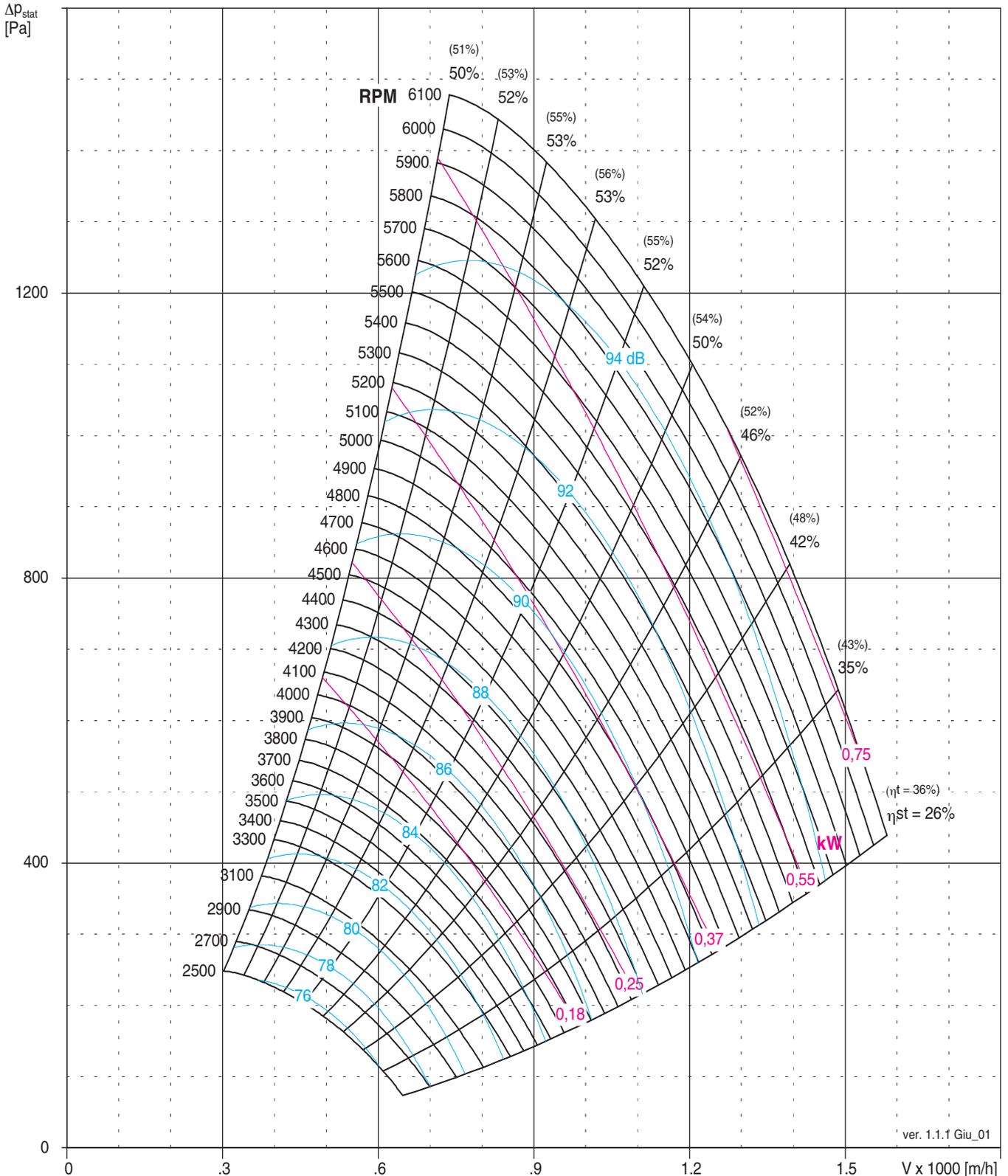
$$P_w = P_{w\text{corr}} \cdot K_p = 1,827 \cdot 0,85 = 1,55 \text{ kW}$$

5. Performance charts	Page
5.1. TE 180	12
5.2. TE 200	13
5.3. TE 225	14
5.4. TE 250	15
5.5. TE 280	16
5.6. TE 315	17
5.7. TE 355	18
5.8. TE 400	19
5.9. TE 450	20



Model/Size	TE 180
Wheel Max RPM [min ⁻¹]	6050
Wheel Max kW	0,8
Wheel No. Blades	8

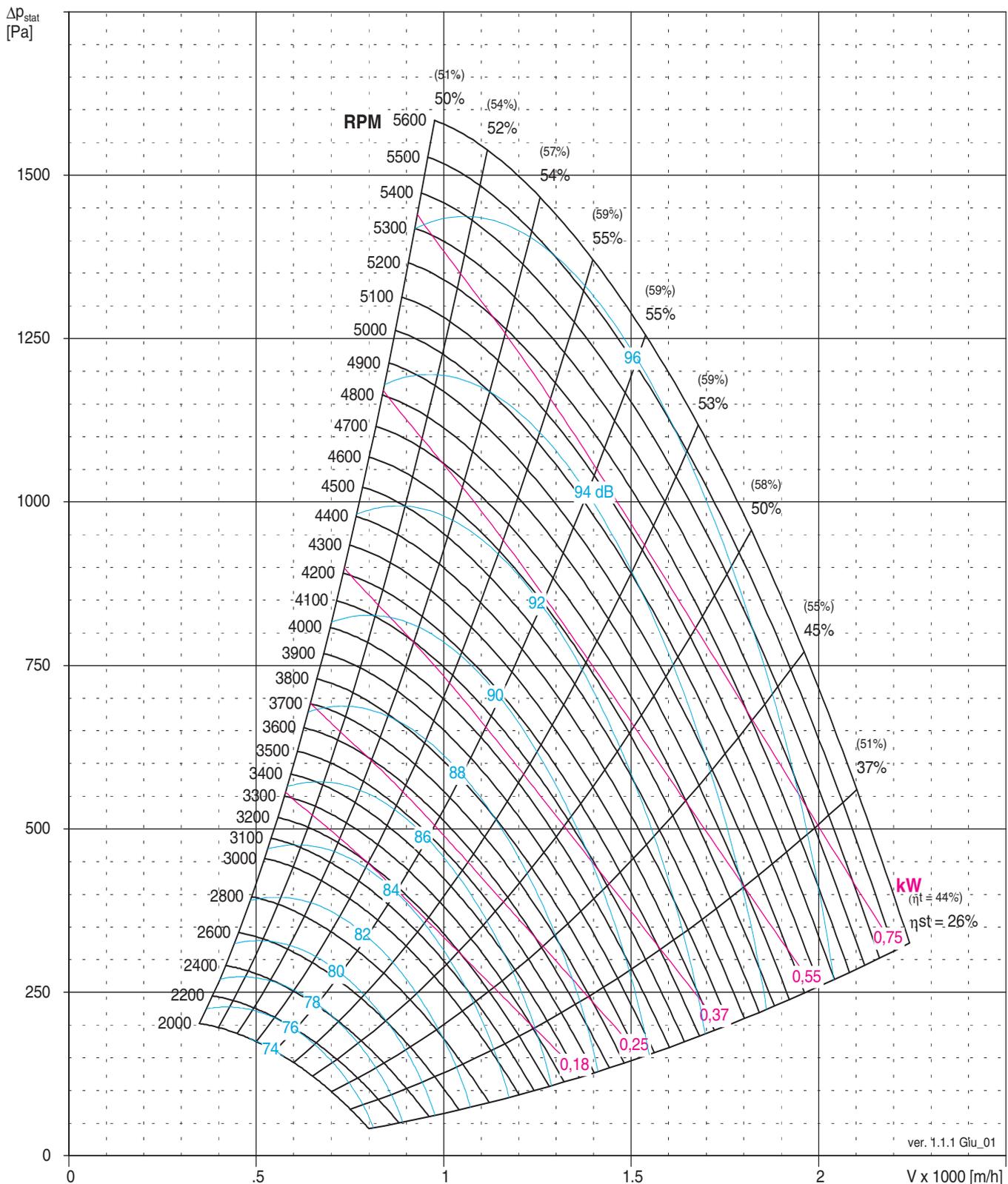
Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{ws} : Total Sound Power Level at the wheel outlet





Model/Size	TE 200
Wheel Max RPM [min^{-1}]	5500
Wheel Max kW	1
Wheel No. Blades	8

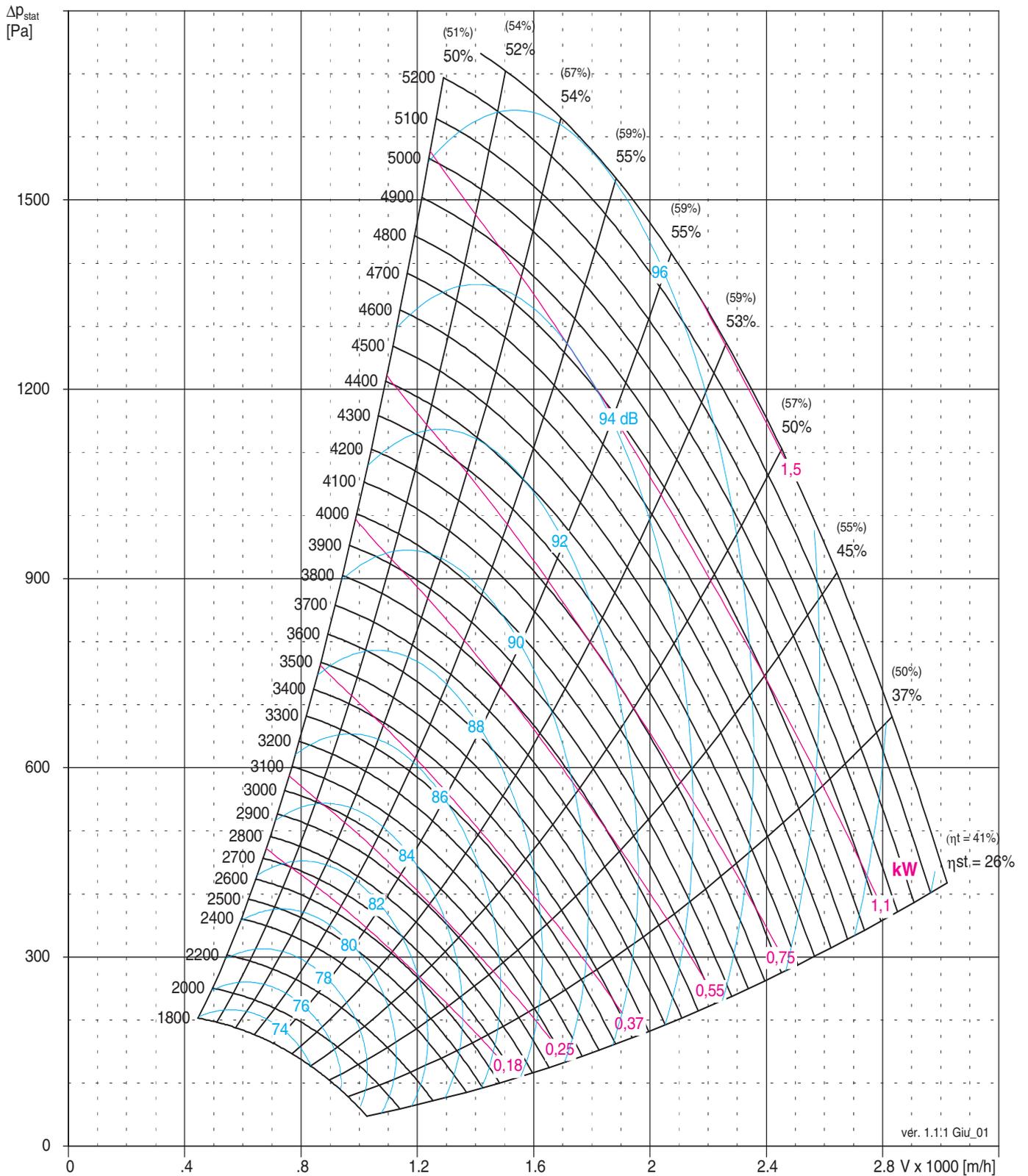
Fan curves plotted for air density: $1,2 \text{ kg/m}^3$
 Free inlet - Free outlet
 L_{we} : Total Sound Power Level at the wheel outlet





Model/Size	TE 225
Wheel Max RPM [min ⁻¹]	5200
Wheel Max kW	1,5
Wheel No. Blades	8

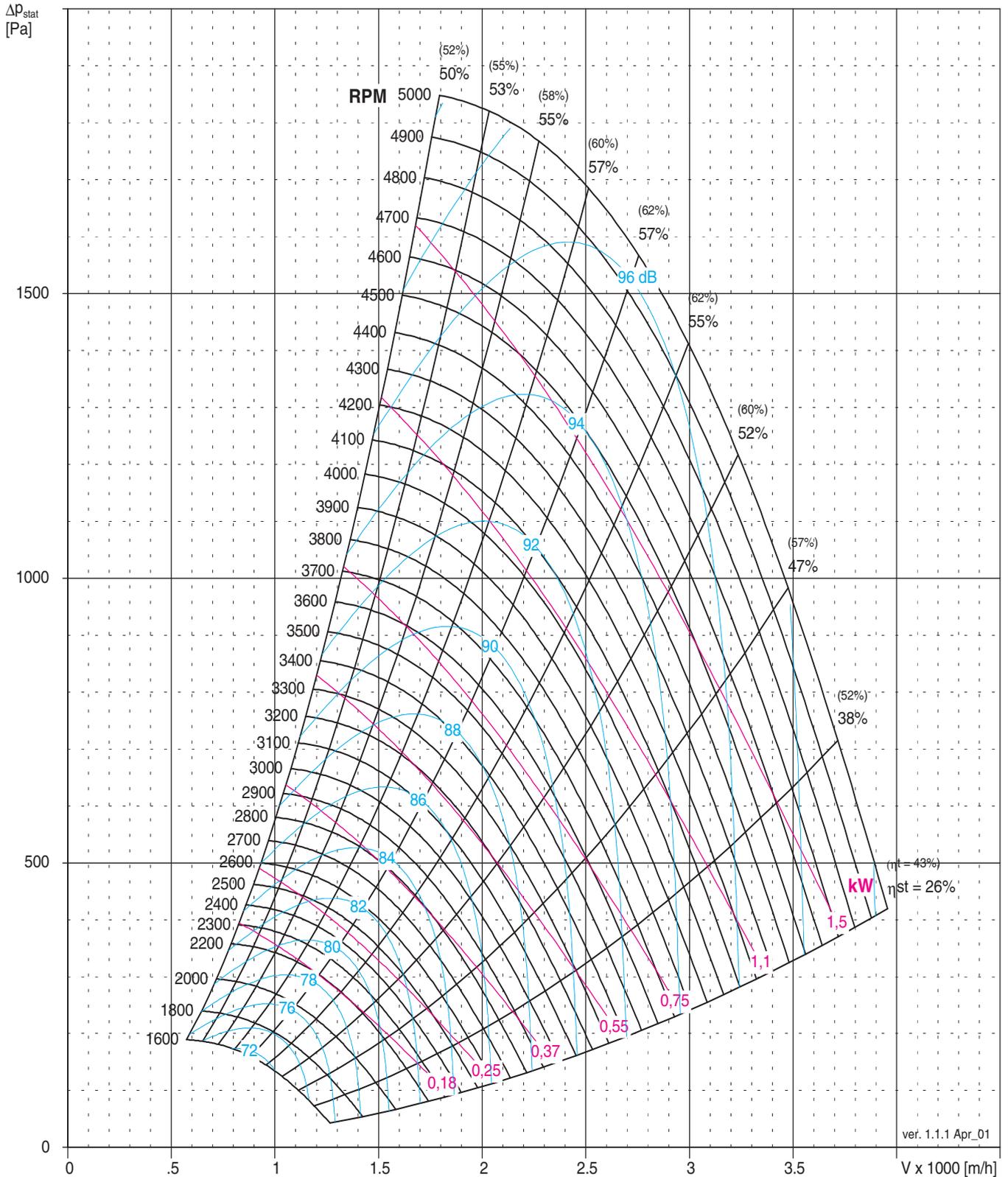
Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{ws} : Total Sound Power Level at the wheel outlet





Model/Size	TE 250
Wheel Max RPM [min ⁻¹]	4900
Wheel Max kW	2,25
Wheel No. Blades	8

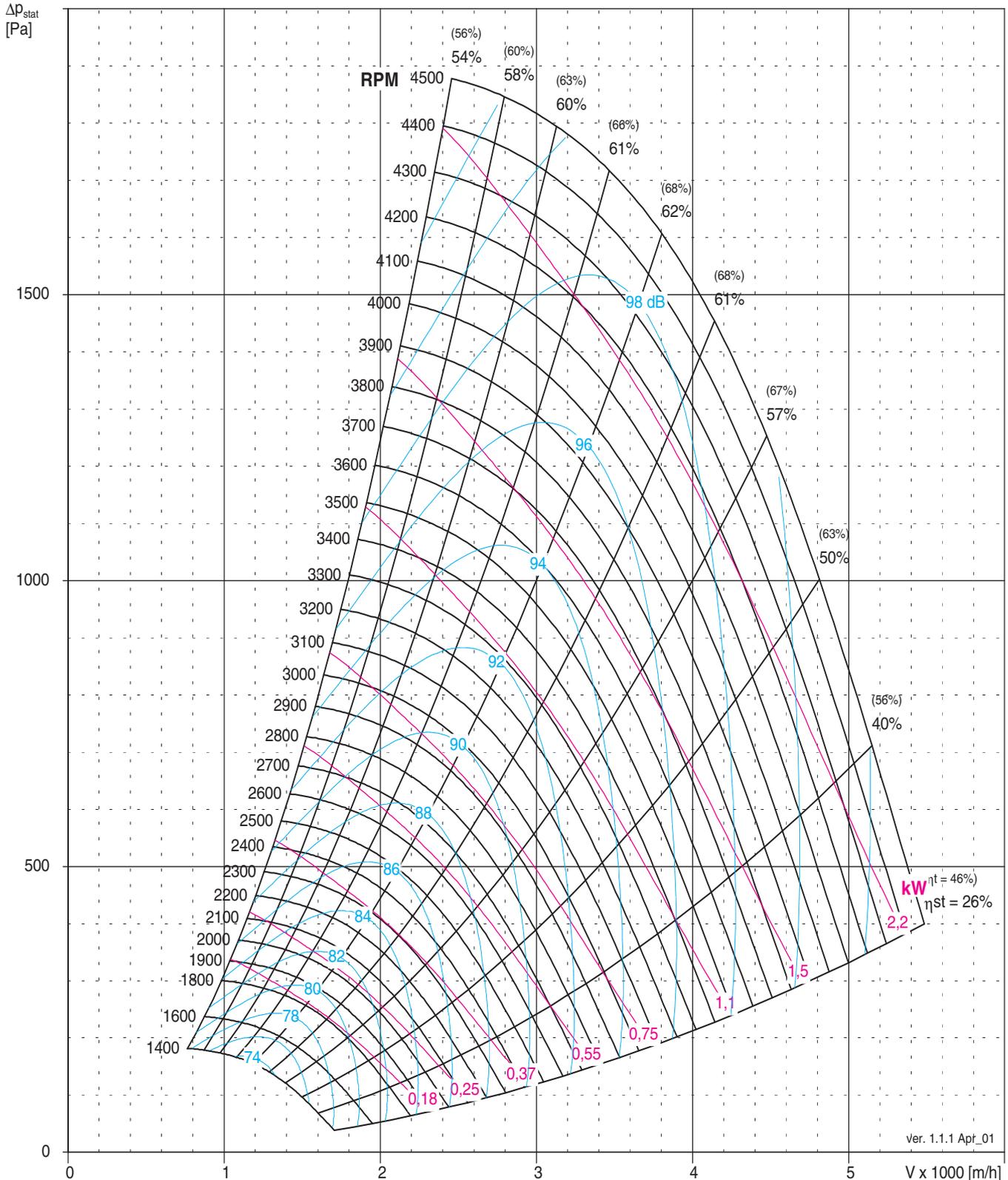
Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{we} : Total Sound Power Level at the wheel outlet





Model/Size	TE 280
Wheel Max RPM [min ⁻¹]	4400
Wheel Max kW	2,75
Wheel No. Blades	8

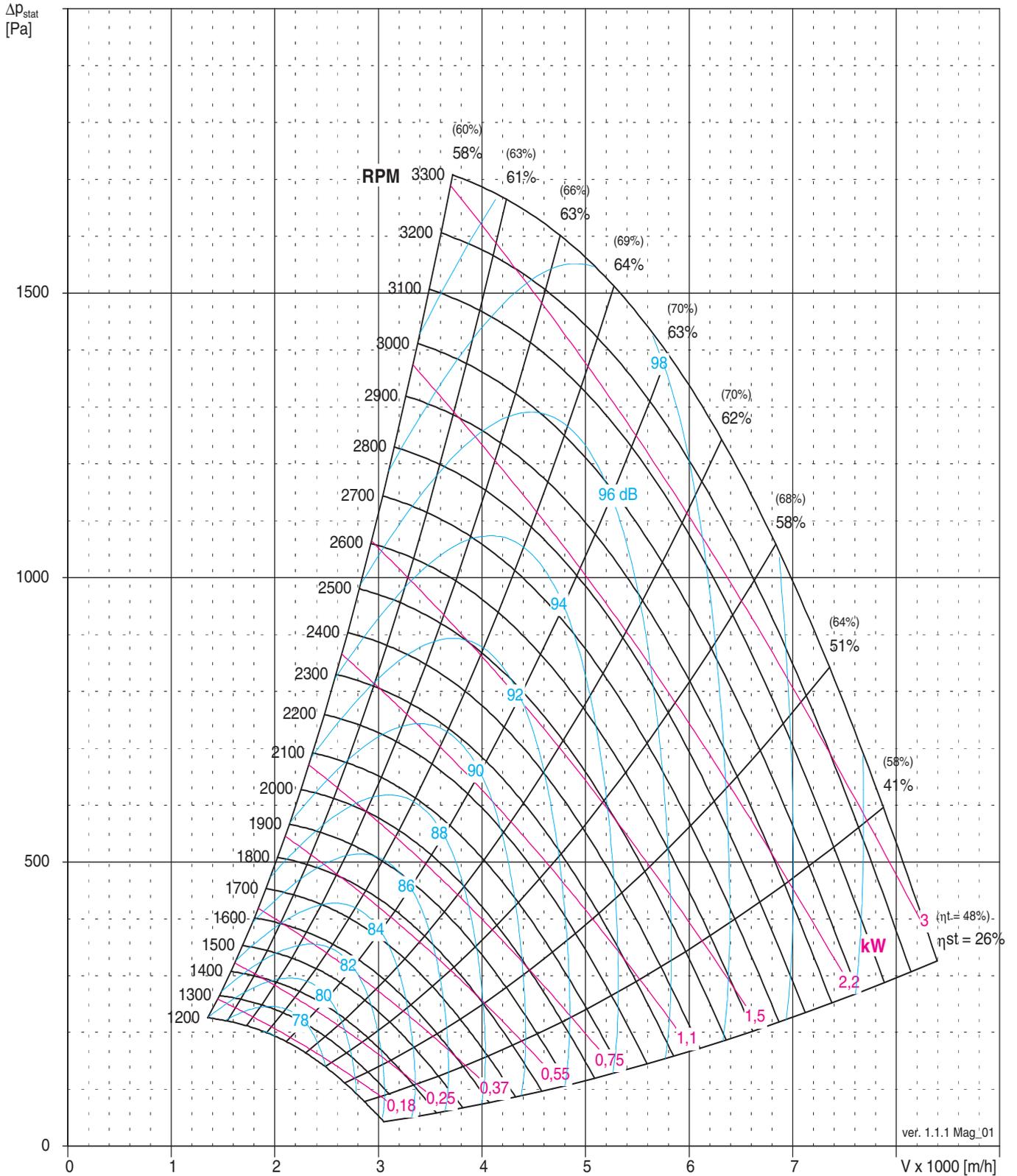
Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{ws} : Total Sound Power Level at the wheel outlet





Model/Size	TE 355
Wheel Max RPM [min ⁻¹]	3200
Wheel Max kW	3,5
Wheel No. Blades	8

Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{ws} : Total Sound Power Level at the wheel outlet

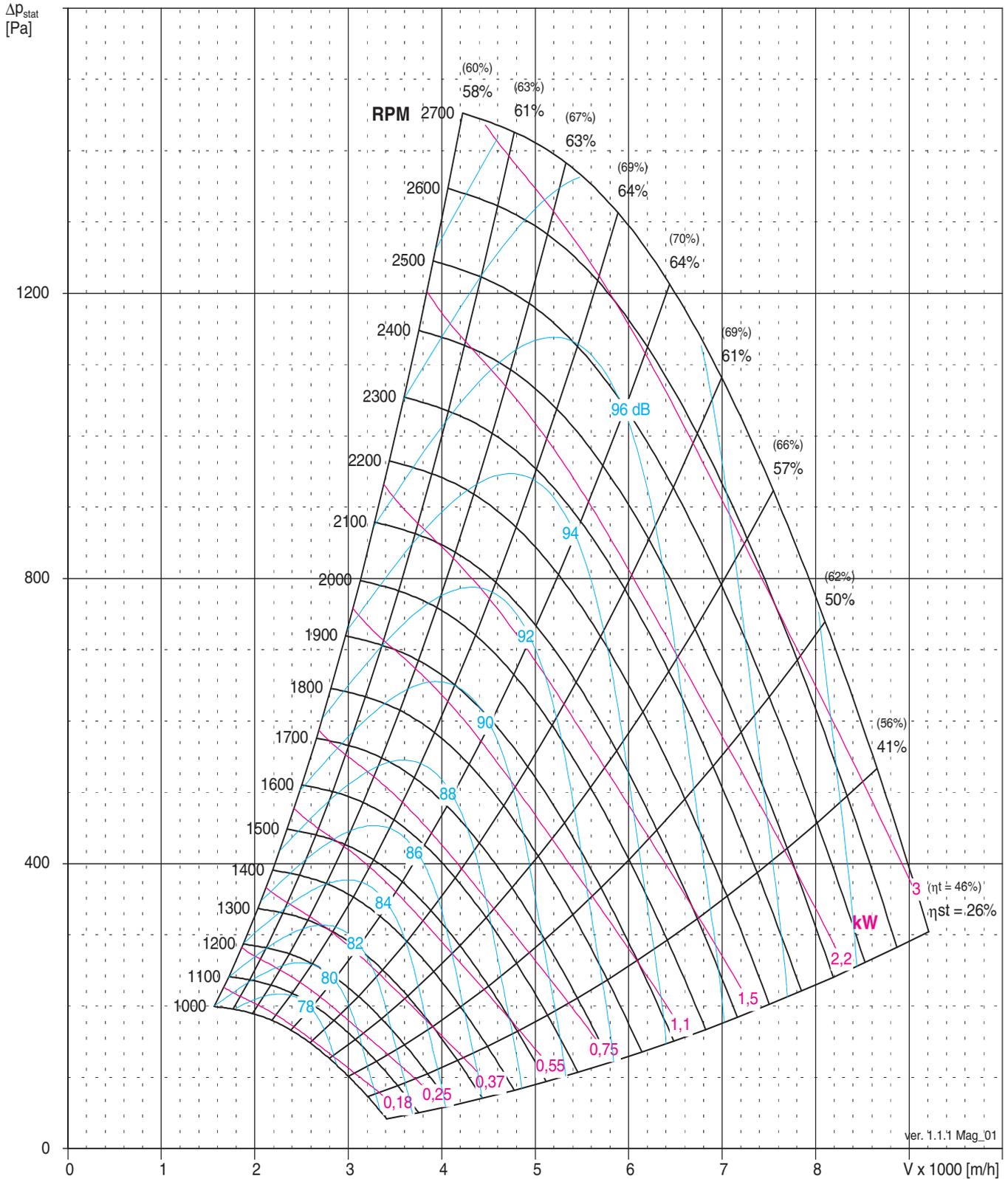


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Model/Size	TE 400
Wheel Max RPM [min ⁻¹]	2650
Wheel Max kW	3,75
Wheel No. Blades	8

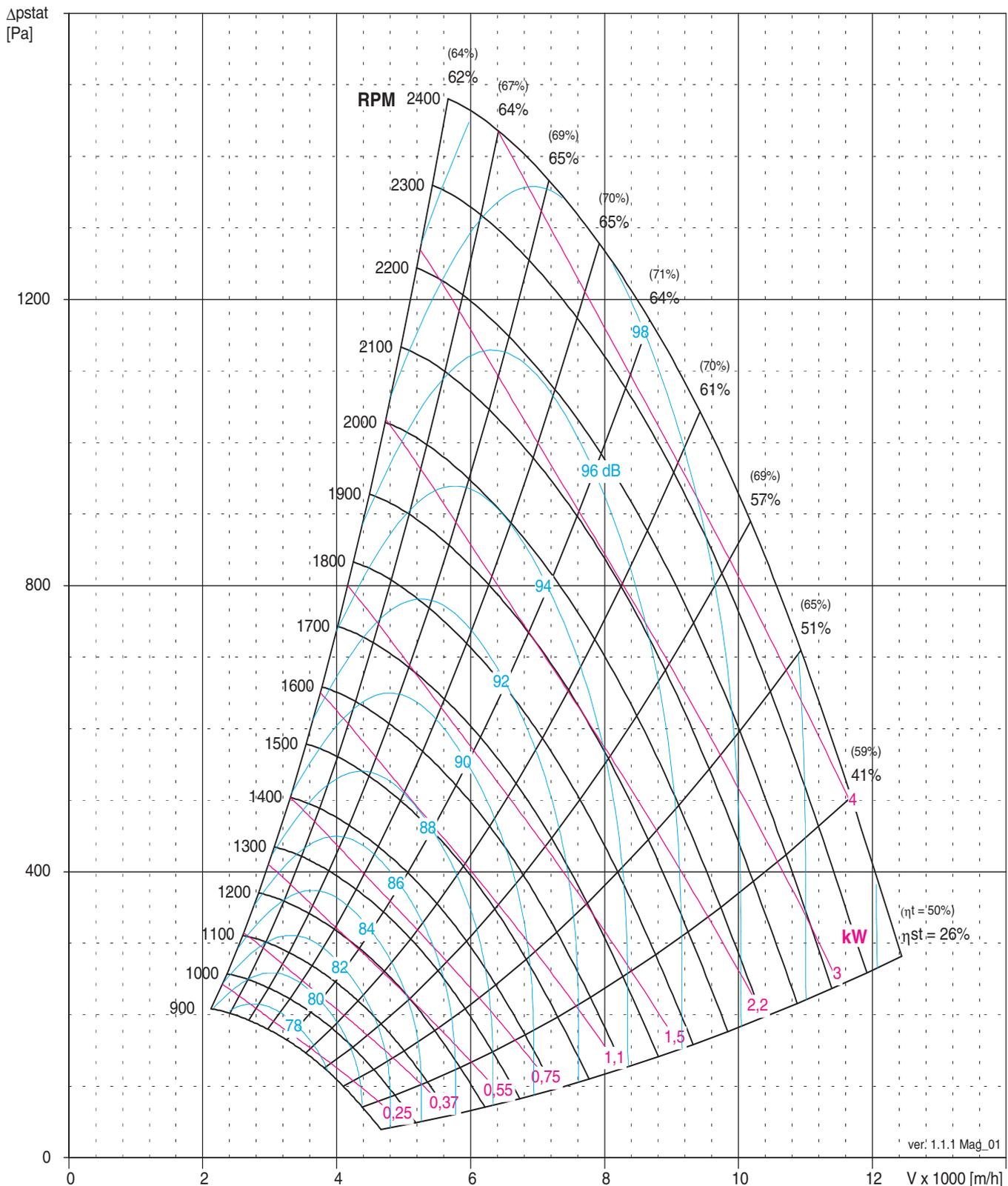
Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{ws} : Total Sound Power Level at the wheel outlet





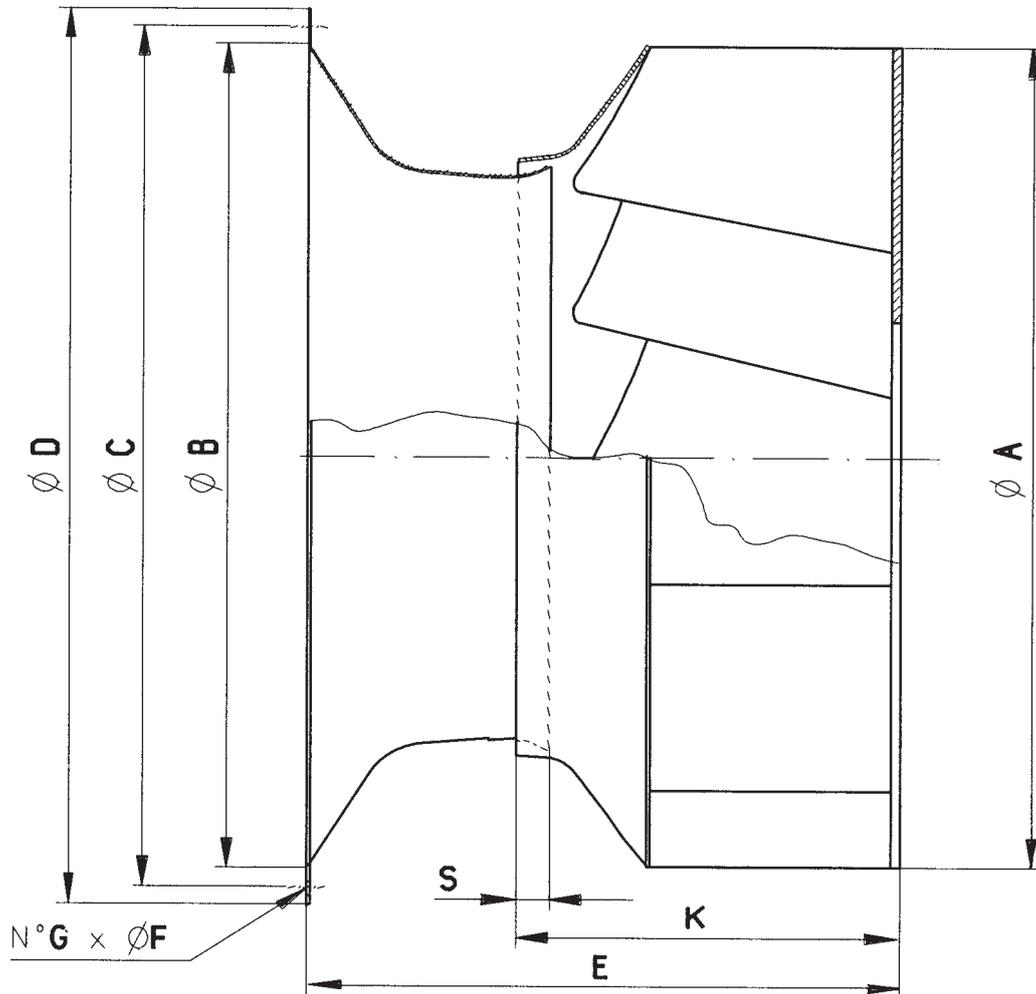
Model/Size	TE 450
Wheel Max RPM [min ⁻¹]	2350
Wheel Max kW	4,5
Wheel No. Blades	8

Fan curves plotted for air density: 1,2 kg/m³
 Free inlet - Free outlet
 L_{ws} : Total Sound Power Level at the wheel outlet





6. WHEEL DIMENSIONS: TE 180 to 450



	ΦA [mm]	ΦB [mm]	ΦC [mm]	ΦD [mm]	E [mm]	ΦF [mm]	G	K [mm]	S [mm]	(*) WHEEL WEIGHT [kg]	INLET CONE WEIGHT [kg]
180	180	167	202	222	116	7,5	6	87	6	0,3	0,25
200	202	181	232	252	129	7,5	6	92	6	0,5	0,35
225	226	211	257	277	145	7,5	6	106	7	0,6	0,45
250	250	228	283	303	165	7,5	6	117	8	0,8	0,6
280	282	262	320	350	188	10	6	131	9	1,1	0,8
315	316	301	355	385	204	10	6	145	11	1,5	1,1
355	357	347	395	425	234	10	8	164	14	2	1,2
400	402	383	440	470	254	10	8	179	14	2,1	1,5
450	455	430	490	530	285	12	8	201	16	3,4	2,6

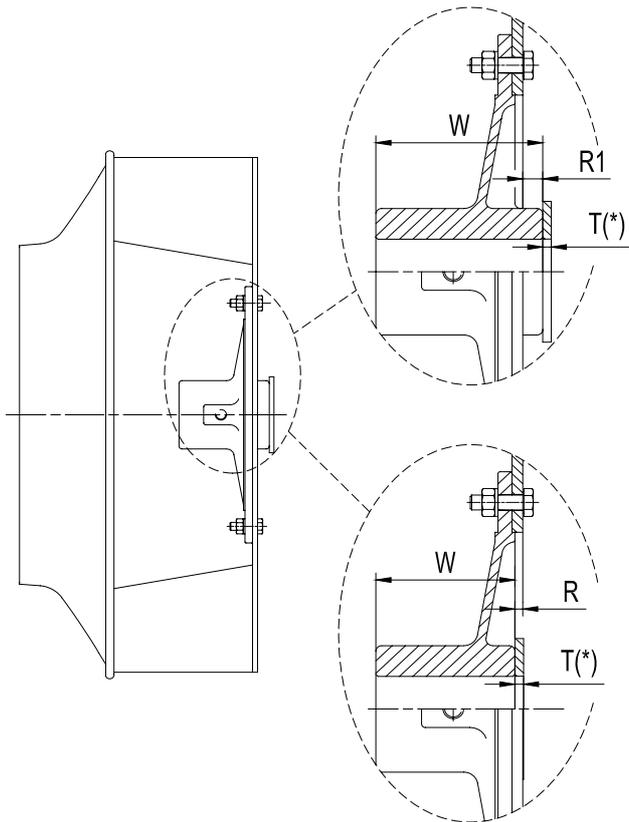
(*) WITHOUT HUB; HUB WEIGHT AND TOTAL WHEEL MOMENT OF INERTIA CAN BE FOUND ON SECTION 7.



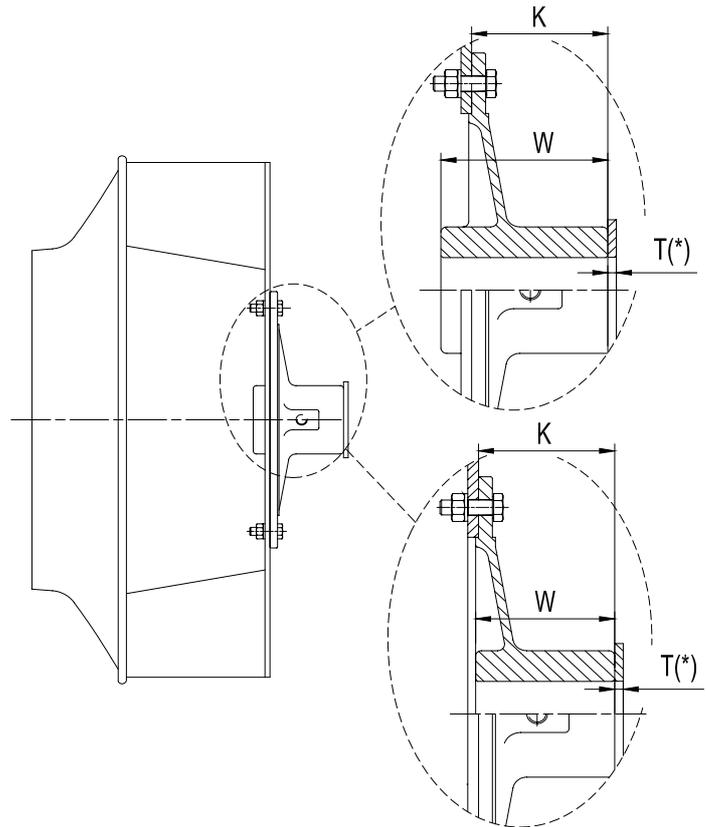
7. MOTOR SIZE FOR DIRECT DRIVEN PLUG FAN

7.1. Hub Arrangements: Internal - HI ; External - HE

WHEEL WITH
ALUMINIUM HUB



WHEEL WITH
ALUMINIUM HUB



(*) steel distance bush to insert between aluminium hub and motor shaft abutting

WHEEL ϕ	MOTOR SIZE	OUTPUT POWER [Kw]	POLES	SHAFT DIAM. [mm]	HUB Weight [kg]	J ** [kg m ²]	W	T	Internal - HI		External - HE		
									R	R1	K		
180	63 A / B	0,18 / 0,25	2	11	0,26	0,033	44	1,5	5		43		
	71 A / B	0,37 / 0,55		14	0,25								
	80 A / B	0,75 / 1,1		19	0,31							0,035	54
200	63 A / B	0,18 / 0,25	2	11	0,26	0,04	44	1,5	5		43		
	71 A / B	0,37 / 0,55		14	0,25								
	80 A / B	0,75 / 1,1		19	0,31							0,05	54
225	63 A / B	0,18 / 0,25	2	11	0,26	0,074	44	1,5	5		43		
	71 A / B	0,37 / 0,55		14	0,25								
	80 A / B	0,75 / 1,1		19	0,31	0,078						54	5
	90 S	1,5		24	0,28								

(**) TOTAL WHEEL MOMENT OF INERTIA (WHEEL+HUB)

SUBJECT TO CHANGE BY COMEFRI



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**GLASS REINFORCED POLYAMID WHEELS
FOR PLENUM FANS - TE**

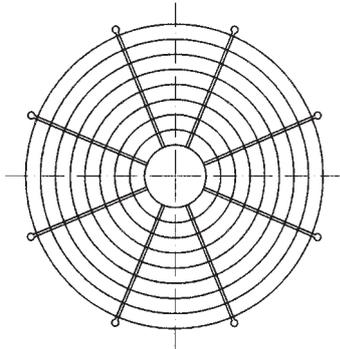
WHEEL ϕ	MOTOR SIZE	OUTPUT POWER [Kw]	POLES	SHAFT DIAM. [mm]	HUB Weight [kg]	J ** [kg m ²]	W	T	Internal - HI		External - HE	
									R	R1	K	
250	63 A / B	0,18 / 0,25	2	11	0,26	0,011	44	1,5	5		43	
	71 A / B	0,37 / 0,55		14	0,25							
	80 A / B	0,75 / 1,1		19	0,31	0,0115	54			5		
	90 S / L	1,5 / 2,2		24	0,28							
280	71 A / B	0,37 / 0,55	2	14	0,25	0,018	44	1,5	5		43	
	80 A / B	0,75 / 1,1		19	0,31		54					5
	90 S / L	1,5 / 2,2		24	0,28							
	100 L	3		28	0,26							
315	71 A / B	0,25 / 0,37	4	14	0,54	0,036	64	3		5	51,5	
	80 A / B	0,55 / 0,75		19	0,52							
	90 S / L	1,1 / 1,5		24	0,49							
	71 A / B	0,37 / 0,55	2	14	0,54							
	80 A / B	0,75 / 1,1		19	0,52							
	90 S / L	1,5 / 2,2		24	0,49							
	100 L	3		28	0,46							
				28	0,46							
355	71 A / B	0,25 / 0,37	4	14	0,54	0,052	64	3		5	51,5	
	80 A / B	0,55 / 0,75		19	0,52							
	90 S / L	1,1 / 1,5		24	0,49							
	71 A / B	0,37 / 0,55	2	14	0,54							
	80 A / B	0,75 / 1,1		19	0,52							
	90 S / L	1,5 / 2,2		24	0,49							
	100 L	3		28	0,46							
	112 M	4		28	0,46							
400	80 A / B	0,55 / 0,75	4	19	0,52	0,08	64	3		5	51,5	
	90 S / L	1,1 / 1,5		24	0,49							
	100 LA / LB	2,2 / 3		28	0,46							
	112 M	4		28	0,46							
	80 A / B	0,75 / 1,1	2	19	0,52							
	90 S / L	1,5 / 2,2		24	0,49							
	100 L	3		28	0,46							
	112 M	4		28	0,46							
450	80 A / B	0,55 / 0,75	4	19	1,21	0,15	54	3		8	38,5	
	90 S / L	1,1 / 1,5		24	1,18		84				68,5	
	100 LA / LB	2,2 / 3		28	1,4							
	112 M	4		28	1,4							
	80 A / B	0,75 / 1,1	2	19	1,21		54				38,5	
	90 S / L	1,5 / 2,2		24	1,18		84				68,5	
	100 L	3		28	1,4							
	112 M	4		28	1,4							

(**) TOTAL WHEEL MOMENT OF INERTIA (WHEEL+HUB)

SUBJECT TO CHANGE BY COMEFRI



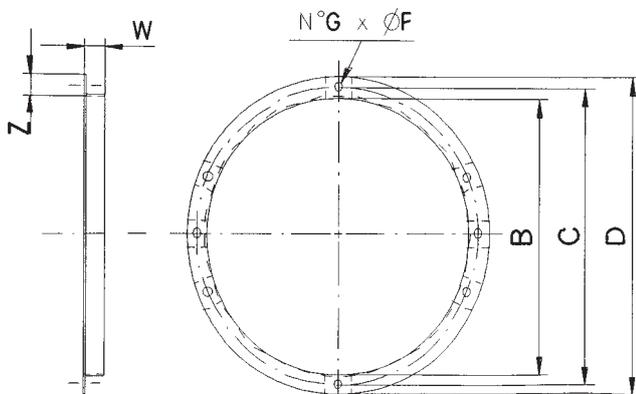
8. ACCESSORIES



8.1. Inlet guard - ZS

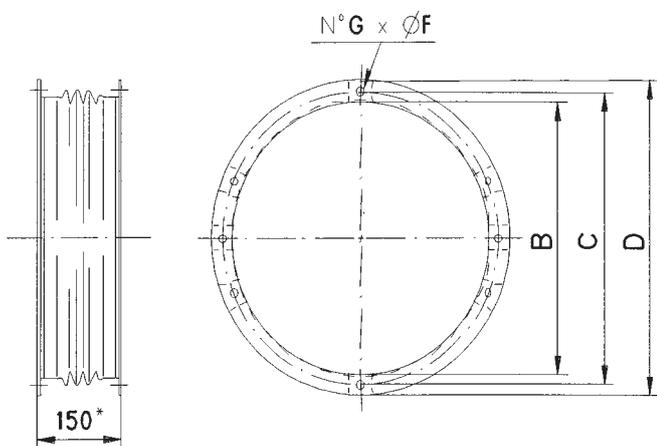
Industrial safety regulations specify that reliable guard must be provided for rotating machine elements.

Inlet and outlet guard are available, according to EN 294



8.2. Inlet Flange - Z

They can be supplied separately or fitted on customer's requirement. From size 180 to 450 they are manufactured in galvanized steel. Their dimensions and drillings are given in table 9.



8.3. Flexible Inlet Connection - ZEL

The flexible connection for the inlet is manufactured with a polyester / PVC fabric and two inlet flanges - Z

Their dimensions and drillings are given in table 9.

	B [mm]	C [mm]	D [mm]	F [mm]	G [mm]	W [mm]	Z [mm]
180	183	202	215	7,5	6	25	16
200	205	232	250	7,5	6	25	23
225	229	257	279	7,5	6	25	25
250	256	283	306	7,5	6	25	25
280	288	320	348	10	6	30	30
315	322	355	382	10	6	30	30
355	361	395	421	10	8	30	30
400	404	440	464	10	8	30	30
450	453	490	513	12	8	30	30

Table 9



9. FLOW METER

The TE wheels can be fitted with a device for measuring and controlling the air volume with a 10% tolerance (normal operating conditions).

In order to establish the accuracy of the measuring device, each TE impeller has been tested in our state of art test laboratory.

The device consist of two or more static pressure measuring points, mounted directly into the inlet cone. The measuring points are manifolded together via a flexible ring tube. Therefore all that is required is for the single tube connected to a pressure measuring device.

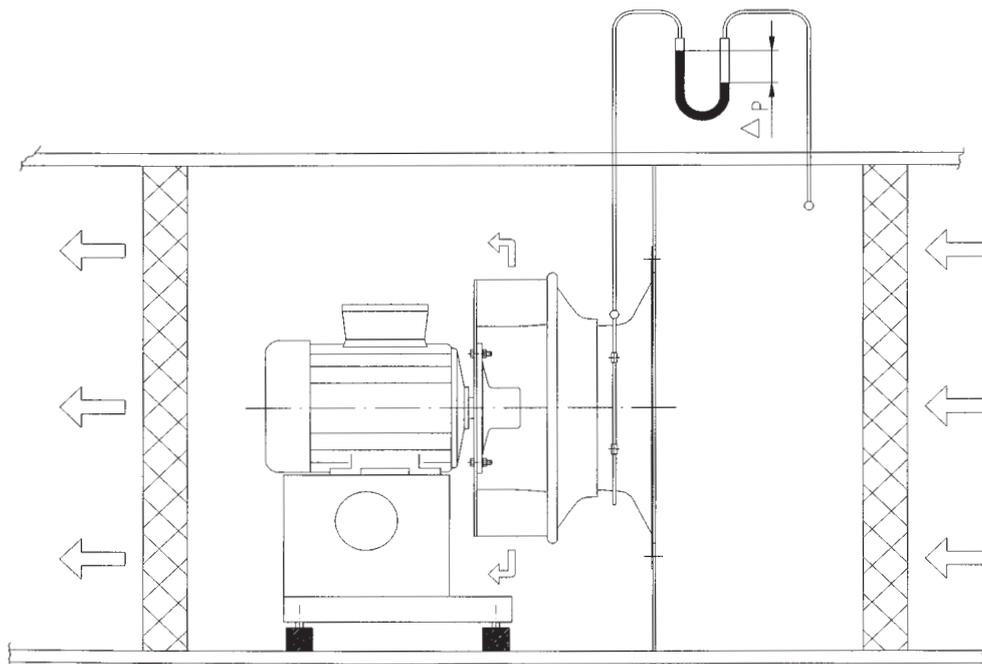
The differential pressure (Δp) measured between the inlet of the AHU and the flow measuring device (see drawing) can be converted to the volume flow by the following formula:

$$V = K \sqrt{\frac{2}{\rho} \cdot (\Delta p)}$$

- V Volume flow [m³/h]
- ρ Air density [kg/m³] (1.2 for standard air)
- Δp Differential pressure [Pa]
- K Calibration factor depending on wheel size

Wheel Size	180	200	225	250	280	315	355	400	450
K	23	30	38	47	59	75	95	123	158

COMEFRI's TE wheels are available with the Flow Meter measuring device all ready factory installed, otherwise a Flow Meter Kit device can be delivered.



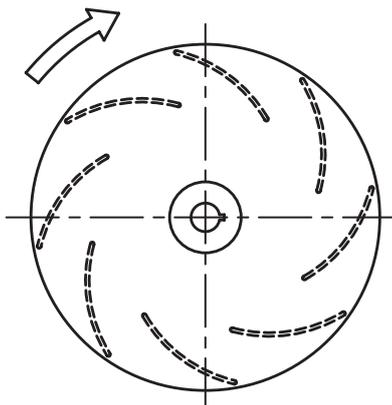


10. ROTATION

The wheel direction of rotation, seen from drive side is:

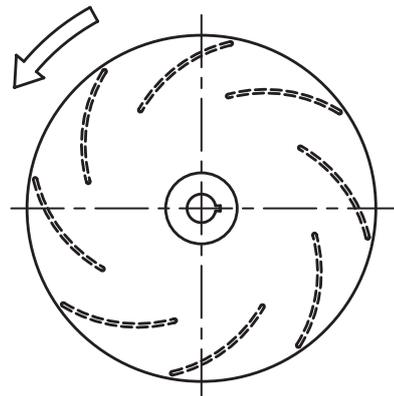
- a) clockwise, if indicated with the symbol RD, or
- b) counter-clockwise if indicated with the symbol LG

RD – CLOCKWISE



View Drive Side

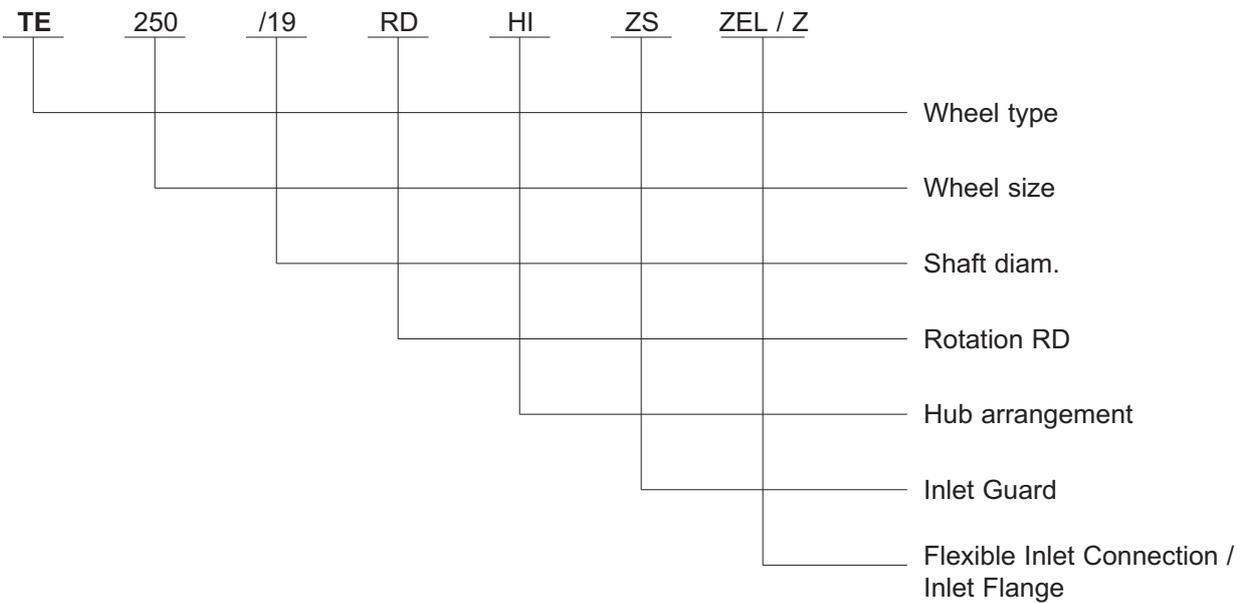
LG - COUNTER CLOCKWISE



View Drive Side

11. PRODUCT IDENTIFICATION

11.1. Wheel reference Code / Example





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**GLASS REINFORCED POLYAMID WHEELS
FOR PLENUM FANS - TE**

Notes

A series of horizontal dotted lines for taking notes.



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**GLASS REINFORCED POLYAMID WHEELS
FOR PLENUM FANS - TE**

We reserve the right to modify fan designs or dimensions in order to enhance our products.

Comefri SpA

Via Buja, 3
I-33010 Magnano in Riviera (UD)
Italy
Tel. +39-0432-798811
Fax +39-0432-783378
www.comefri.com
E-mail: info@comefri.com

Comefri UK Ltd

Carters Lane, 8 Kiln Farm
Milton Keynes, MK11 3 ER
Great Britain
Tel. +44-1908-56 94 69
Fax +44-1908-56 75 66
www.comefri.com
E-mail: sales@comefri.co.uk

Comefri GmbH

Dieselstrasse 4
84051 Essenbach-Altheim
Germany
Tel. +49-8703-93 20 0
Fax +49-8703-93 20 40
www.comefri.de
E-mail: info@comefri.de

Comefri Nordisk ApS

Mileparken, 18
DK 2740 Skovlunde
Denmark
Tel. +45-44-92 76 00
Fax +45-44-92 55 33
www.comefri.com
E-mail: mail.dk@comefri.com

Comefri France S.A.

10, Rue des Frères Lumière
69740 Genas
France
Tel. +33-4-72 79 03 80
Fax +33-4-78 90 69 73
www.comefri.com
E-mail: info@comefrifrance.fr

Comefri USA, Inc

330 Bill Bryan Boulevard
Hopkinsville, KY 42240
USA
Tel. +1-270-881-1444
Fax +1-270-889-0309
www.comefriusa.com
E-mail: sales@comefriusa.com

